

OPTIMIZATION OF HYDRAULIC PARAMETERS OF RETICULATED AGITATORS FOR DEMINERALIZATION PLANTS

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In recent years, active measures have been taken for broad-scale exploitation of oil and gas reserves within continental shelves. Not excluded is the fact that the demand for demineralization of sea water for residential and commercial needs has increased significantly over time in these regions.

At the present time, woven grids are widely used in hydraulic and aerodynamic systems [1–4] that control the work of liquid and gas media for subsequent physicochemical transformations (electrochemical, catalytic, ion-exchange, thermal). A disadvantage of the grids employed is their simple structure, which does not create conditions that permit control of liquid-flow agitation during execution of the indicated processes. Agitation leads to activation of a flow effect on the near-boundary layer, and to a reduction in the clearance dimensions of the plant.

The possibility of controlling the hydraulic and aerodynamic parameters of three-dimensional break-through grids over a broad range supporting optimal operating conditions has been ascertained on the basis of many years' experience with these grids [5–7]. Grids fabricated by the waste-free break-through/draw-out method offers a number of advantages over woven grids in terms of hydraulic parameters. They are universal owing to the broad possibility of varying the three directions of motion of the front of the liquid medium, which ensures a controllable flow regime. Moreover, they can be fashioned from any moldable metallic and polymeric materials.

Let us examine the hydraulic and aerodynamic characteristics of three-dimensional break-through grids, which are required to evaluate the effect on their linear parameters as assigned by automated machinery. Chuvpilo [7] cites three possible effects of a rhombic break-through grid on the movement of the front of the liquid medium flowing within its channels (I – across the smooth plane of the grid; II – along the smooth plane of the grid and perpendicular to the small diagonal of the rhombus; and, III – along the smooth plane of the grid and perpendicular to the large diagonal of the rhombus), and their interrelation for the free (A), deflecting (B), and blind (C) zones of the channel section.

The hydraulic resistance of a break-through grid for separator-agitators of electro dialysis demineralization plants can be calculated by treating the grid as bundles of parallel tubes:

$$\Delta p = \lambda \frac{L}{d_{\text{eq}}} \frac{\rho_0^2}{\rho_m} \frac{u_0^2}{2},$$

where Δp is the head loss over a length L , λ is the coefficient of hydraulic resistance of the break-through grid, which takes into account the effect of the moving front of the medium, ρ_0 is the density of the medium in the inlet section, u_0 is the average velocity of the flow proceeding into the inlet section, d_{eq} is the equivalent diameter of the grid [see expression (1) below], and ρ_m is the average density of the medium at the inlet and outlet (in the special case, $\rho_0 = \rho_m$).

The following experimental relationships are used for laminar and turbulent flow regimes of the medium:

$$\lambda = A_l / \text{Re}; \quad \lambda = A_t / \text{Re}^m,$$

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where A_l and A_t are experimental coefficients for the laminar and turbulent flow regimes of the medium, Re is the Reynolds number, and m is an exponent.

Table 1 lists parameters for two types of waste-free break-through rigid-vinyl-plastic grids (Fig. 1) [type 1 is manufactured by the Expanded Metal and K^o Company (Great Britain), while type 2 is produced on an automated ABS-1 machine (Russia)] for separators-agitators in electro dialysis demineralization plants.

Considering the variable structure of the break-through grid, the equivalent diameter d_{eq} , which enters into the Reynolds number, must be determined for the hydraulic calculations:

$$d_{eq} = \frac{4f_{eq}}{\chi_{eq}},$$

or

$$d_{eq} = \frac{2(1 - K_v)(2\tau \sin \gamma + \delta \cos \gamma)}{1 + 0.5K_k}, \quad (1)$$

where

$$K_v = \frac{\tau \delta}{t(2\tau \sin \gamma + \delta \cos \gamma)};$$

$$K_k = \frac{2(\delta + \tau) - \delta k / T}{t}.$$

The equivalent wetted perimeter should be computed for all three moving fronts of the medium:

$$\chi_{eq}^I = \frac{Tt(2 + K_k)}{2\tau \sin \gamma + \delta \cos \gamma};$$

$$\chi_{eq}^{II} = t(2 + K_k);$$

$$\chi_{eq}^{III} = T(2 + K_k).$$

The equivalent sectional area of the channel is also calculated for the three moving fronts of the medium:

$$f_{eq}^I = Tt(1 - K_v);$$

$$f_{eq}^{II} = t(1 - K_v)(2\tau \sin \gamma + \delta \cos \gamma);$$

$$f_{eq}^{III} = T(1 - K_v)(2\tau \sin \gamma + \delta \cos \gamma).$$

In certain cases, it is necessary to know the relationship between the total-surface area of the grid and the volume S_{sp}^v (mass S_{sp}^m) of the medium wetting this surface:

$$S_{sp}^v = \frac{K_k}{(1 - K_v)(2\tau \sin \gamma + \delta \cos \gamma)};$$

$$S_{sp}^m = \frac{10^3 K_k}{\rho(1 - K_v)(2\tau \sin \gamma + \delta \cos \gamma)}.$$

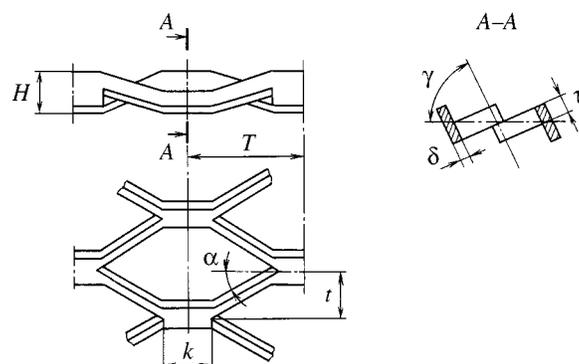


Fig. 1. Diagram showing break-through grid.

TABLE 1

Indicator	Grid	
	Type 1	Type 2
Thickness δ of blank, mm	0.55	0.65
Spacing T of transverse blade feed, mm	3.00	4.25
Spacing τ of blank feed, mm	0.42	0.39
Draw spacing t , mm	1.50	1.77
Grid thickness H , mm	1.00	1.00
Width k of cross piece at connection between grid cells, mm	1.50	2.50
Angle γ of three-dimensional rotation of crosspiece after break-through with respect to plane of blank	60°	60°
Angle α of draw	45°	44°
Grid-opening factor $K_o = 1 - (\tau/t)$	0.72	0.78
Catalyst space factor K_c for grid	1.11	0.97
Concentration factor (ratio of total surface area of waste-free grid to area of catalyst plane) K_v	0.154	0.143
Specific surface area of grid		
with respect to volume S_{sp}^v , mm^{-1}	1.31	1.13
with respect to mass S_{sp}^m , mm^2/g	1310	1130
Equivalent area of flow section, mm^2		
f_{eq}^I	3.81	6.45
f_{eq}^{II}	1.27	1.52
f_{eq}^{III}	2.54	3.64
Equivalent wetted perimeter, mm		
χ_{eq}^I	14.00	22.40
χ_{eq}^{II}	4.67	5.26
χ_{eq}^{III}	9.35	12.60
Equivalent diameter of grid, mm		
d_{eq} from formula (1)	1.09	1.15
experimental d_{eq}^{ex}	1.05	1.14

TABLE 2

Item No.	Moving front of medium	Computed coefficient	Grid	Moving front of medium						
				I		II		III		
				λ_A	λ_B	λ_A	λ_B	λ_A	λ_B	λ_C
1	I	λ_A	Type 1	1	2.34	1.45	1.35	2.80	2.80	1.40
2		λ_B		0.43	1	0.61	0.57	1.20	1.20	0.60
3	II	λ_A	Type 1	0.69	1.64	1	0.95	1.94	1.94	0.97
4		λ_B		0.74	1.75	1.05	1	2.04	2.04	1.02
5	III	λ_A	Type 1	0.36	0.84	0.52	0.49	1	1	0.50
6		λ_B		0.36	0.84	0.52	0.49	1	1	0.50
7		λ_C		0.72	1.66	1.03	0.98	2.00	2.00	1
8	I	λ_A	Type 2	1	2.72	1.41	1.52	4.60	4.60	1.78
9		λ_B		0.37	1	0.52	0.56	1.24	1.24	0.48
10	II	λ_A	Type 2	0.71	1.92	1	1.07	2.37	2.37	0.92
11		λ_B		0.66	1.78	0.94	1	2.20	2.20	0.85
12	III	λ_A	Type 2	0.22	0.81	0.42	0.45	1	1	0.39
13		λ_B		0.22	0.81	0.42	0.45	1	1	0.39
14		λ_C		0.56	2.08	1.08	1.18	2.55	2.55	1

Table 2 presents computed coefficients λ_A , λ_B , and λ_C in zones A, B, and C, respectively, for an electro dialysis demineralization plant, which facilitate determination of conditions for medium conversion from one state to another for control of processes involving physicochemical transformations. For example, the number 1.94 (in the third row) indicates that conditions for the flow of liquid for the grid manufactured by the Expanded Metal and K^o Company in zone A of moving front II of the medium are 1.94 times better than those in zone A of moving front III of the medium.

In a similar case for the grid fabricated on the automated ABS-1 machine, the number 2.37 (tenth row in Table 2) indicates better conditions for conversion to a turbulent regime while retaining more favorable conditions for flow resistance, which are characterized by relationships between the deflecting and blind zones (the numbers 0.50 and 0.39 in the fifth and twelfth rows of Table 2).

The following values of the coefficient λ_A^{II} were obtained as a result of the hydraulic investigations of break-through grids for an electro dialysis demineralization plant, which were produced on the automated ABS-1 machine with the parameters indicated in this study:

$$\lambda_A^{\text{II}} = \frac{664}{\text{Re}} \quad \text{for } \text{Re} \leq 10.6$$

for a laminar flow regime, and

$$\lambda_A^{\text{II}} = \frac{302}{\text{Re}^{0.667}} \quad \text{for } \text{Re} \geq 10.6 \quad (2)$$

for a turbulent flow regime.

Values of the coefficient λ_A^{II} for break-through grids fabricated by the Expanded Metal and K^o Company with the parameters cited in this study are as follows:

$$\lambda_A^{\text{II}} = \frac{1660}{\text{Re}} \quad \text{for } \text{Re} \leq 26.5$$

for a laminar flow regime, and

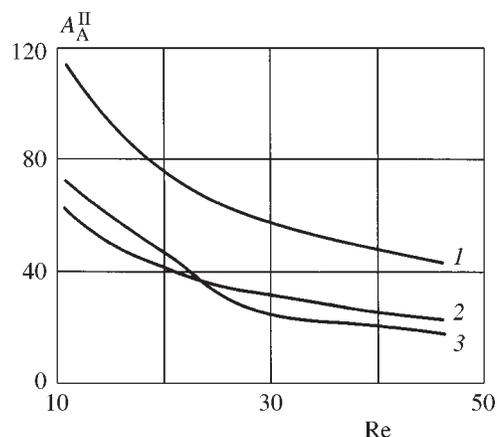


Fig. 2. Dependence of coefficient λ_A^{II} on number Re: 1, 2) from formulas (3) and (2); 3) according to author's data (experiment).

$$\lambda_A^{\text{II}} = \frac{557}{\text{Re}^{0.667}} \quad \text{for } \text{Re} \geq 26.5 \quad (3)$$

for a turbulent flow regime.

Figure 2 shows the effect of Reynolds number on the coefficient λ_A^{II} of hydraulic resistance to movement of the flow.

Analysis of break-through grids produced by the Expanded Metal and K^o Company (curve 1) and grids manufactured on the automated ABS-1 machine indicated that the critical Reynolds number is higher in the first case (26.5). This inevitably led to an increase in the length of the contact (membrane) portion of the apparatus, and the clearance dimensions of the electro dialysis demineralization plant, to greater head losses (by 2.5 times), and, consequently, to an increase in the energy consumed by the circulation pumps.

Results of the hydraulic investigations therefore confirm the computational and experimental relationships cited, which can be used to optimize the operating conditions of electro dialysis demineralization plants employing break-through grids.

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